

Development of an Evaporation Model for HD on Non-Porous Surfaces

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- To develop a simple engineering tool that can predict the evaporation rate of HD on non-porous surfaces and provide information about
 - The amount of mass being evaporated and transported by the wind
 - The amount of mass being absorbed/desorbed into a porous substrate
 - The basic behavior of a drop under the outdoor conditions



- > Evaporation
 - Modeling sessile drop behavior
- Validation
- > Generalization
 - How to generalize our efforts to enhance prediction capability by
 - ✓A hybrid approach
 - ✓ Imposing outdoor conditions
- Linkage to porous substrate



Evaporation Module

- A module that is mostly based on first principles and provides the following information
 - Forcing function the evaporation rate, m
 ✓ Can be modified, improved, replaced, ...
 - Topology of the droplet by solving a differential equation using the forcing function
 - Evaporated mass being added to the atmospheric air
 - Remaining mass to be transported through the porous substrate

Evaporation Module (cont'd) Model Development

- Forcing Function
 - Constant base area for a drop (Model A)

$$\dot{m} = 2\pi C R_s f \frac{\mu}{m} \left(F + C_1 R e^m r^n \right) \ell n \left(1 + B \right) = 2\pi C R_s \frac{\mu}{m} \left(F + C_1 R e^m r^n \right) \ell n \left(1 + B_M \right)$$

$$C = \frac{h}{R}$$

Shrinking base area for a drop (Model B)

$$\dot{m} = 2\pi R_s (1 - \varphi) f \frac{\mu}{m} (F + C_1 Re^m r^n) \ell n (1 + B) \quad or$$

$$\dot{m} = 2\pi R_s (1 - \varphi) f \frac{\mu}{m} (F + C_1 Re^m r^n) \ell n (1 + B_M)$$



➤Interaction among drops – Group Theory

■ Model 1:
$$G_{evap} = \frac{1}{1 + G_c}$$
 with $G_c = \frac{C(F + C_1 \quad {}^m \quad {}^n)}{C(F + C_1 \quad {}^m \quad {}^n)} \quad N^q(\frac{r}{\lambda})$

Where C and q can be determined experimentally. C=3 and q=2/3

Model 2:

$$G_c = \sum_{n=0}^{\infty} \frac{(-1)^n \quad \eta_0}{[(n+1)\eta_0]}$$
Where $\eta_0 = \frac{-1}{R}$ and 2λ represents the distance between two drops

$$\lambda = \frac{1}{2N^p}$$
where N is the number of drops and $p = \frac{1}{3}$

$$G_{evap} = \begin{cases} G_c^q & \text{for } N \ge n \\ \frac{1}{q} G_c^q & \text{for } N < n \end{cases}$$



Evaporation Module (cont'd) Model Development

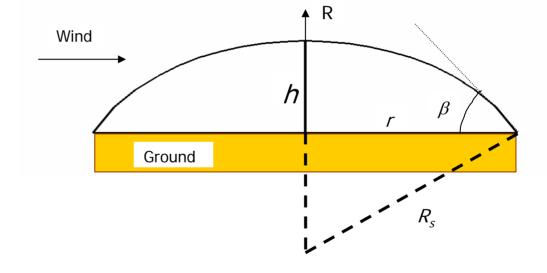
- \triangleright (\dot{m} is a function of time also)
- Model A

$$\frac{dh}{dt} = \frac{-\dot{m}}{\pi \rho_{\ell} h^{2} \left(\frac{3}{C} - 1\right)}$$

$$h/R = C$$

Model B

$$\frac{dh}{dt} = \frac{-\dot{m}}{\rho_{\ell} \frac{\pi}{2} (r^2 + h^2)}$$



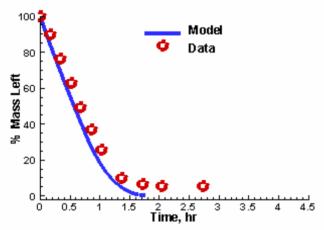


- Validate Model
- Create a matrix for the entire possible domain of operation
- Fill the matrix using the analytical model
- > Use neural network curve-fit
- Create a simple engineering equation for application

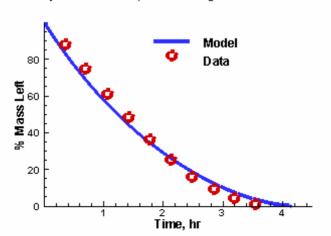


Evaporation Module (cont'd) Validation

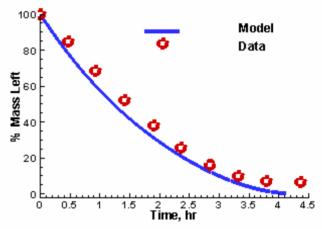
HD on Glass, Wind Velocity = 1.77 m/s, Drop Size = 1 μ L Air Temperature = 35°C, m=1.180 mg



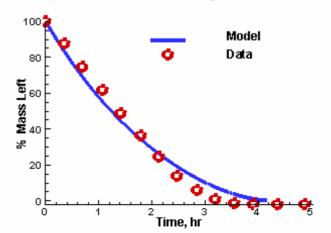
HD on Glass, Wind Velocity = 0.26 m/s, Drop Size = 1 μ L Air Temperature = 35°C, m=1.192 mg



HD on Glass, Wind Velocity = 0.26 m/s, Drop Size = 1 μ L Air Temperature = 35 $^{\circ}$ C, m=1.184 mg



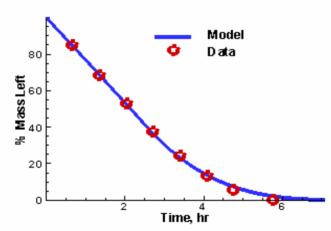
HD on Glass, Wind Velocity =0.26 m/s, Drop Size = 1 μ L Air Temperature = 35°C, m=1.204 mg



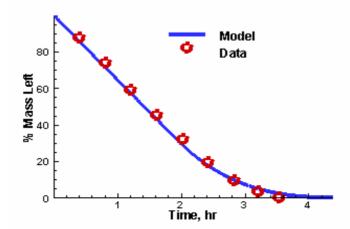


Evaporation Module (cont'd) HD on Non-Porous Surface

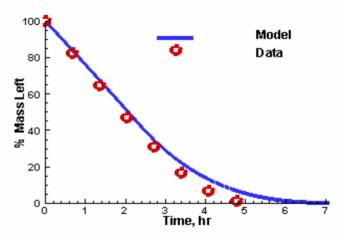
HD on Glass, Wind Velocity = 3.66 m/s, D rop Size = 1 μ L Air Temperature = 15°C, m=1.200mg



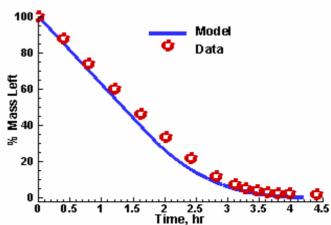
HD on Glass, Wind Velocity = 1.77 m/s, Drop Size = $6 \mu L$ Air Temperature = $35^{\circ}C$, m=6.884 mg



HD on Glass, Wind Velocity = 3.66 m/s, Drop Size = 1 μ L Air Temperature = 15°C, m=1.264 mg



HD on Glass, Wind Velocity = 1.77 m/s, Drop Size = 6 μL Air Temperature = 35°C, m=7.000mg

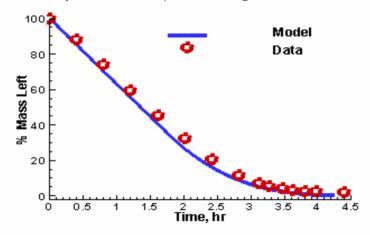




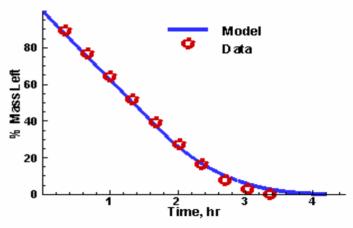
Evaporation Module (cont'd) HD on Non-Porous Surface

HD on Glass, Wind Velocity = 1.77 m/s, Drop Size = 6 μ L Air Temperature = 35°C, m=7.244 mg

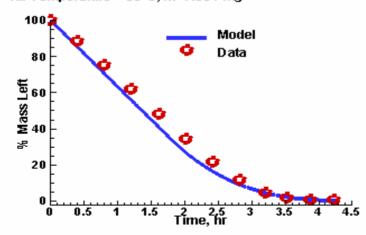
HD on Glass, Wind Velocity = 1.77 m/s, Drop Size = 6 μ L Air Temperature = 35°C, m=7.022 mg

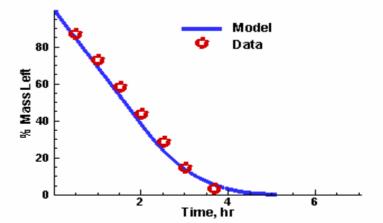


HD on Glass, Wind Velocity = 1.77m/s, Drop Size = 6 μ L Air Temperature = 35°C, m=7.364 mg



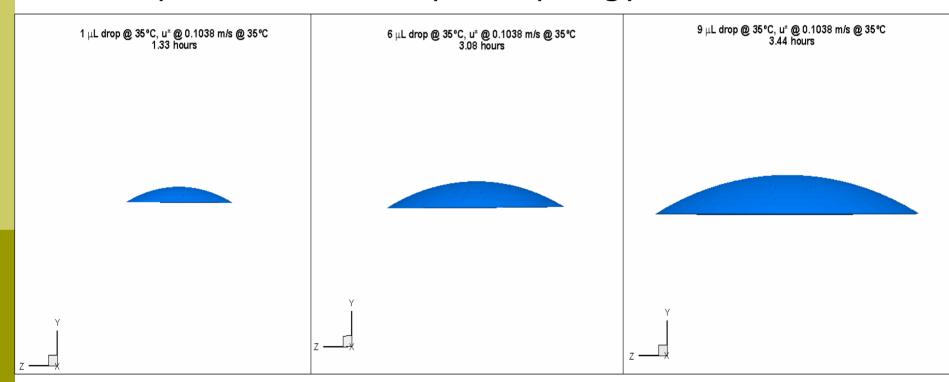
HD on Glass, Wind Velocity = 1.77 m/s, Drop Size = 9 μ L Air Temperature = 35°C, m=12.033 mg





Evaporation Module (cont'd) - HD on Non-Porous Surface - Droplet Topology

Sample animated droplet topology



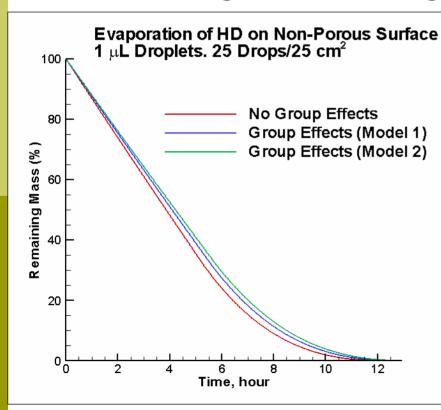
CHIMICAL AND BIOLOGICAL DELICATION

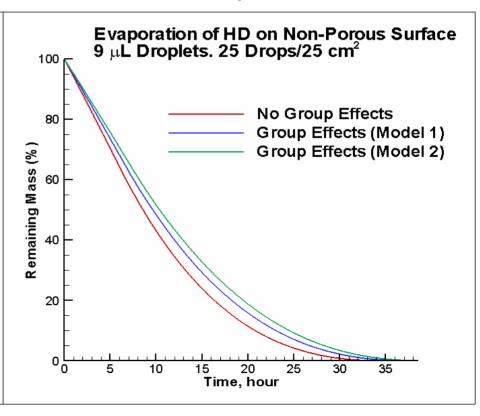
Evaporation Module (cont'd)

HD on a Non-Porous Surface – Group

Theory

- > There are two group models embedded in the code
 - Negligible for small sparse drops
 - More significant for larger and denser drops







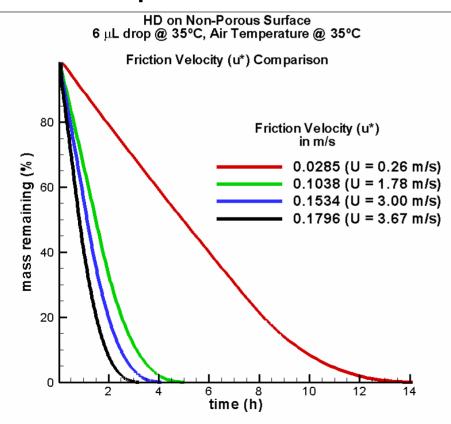
Evaporation Module (cont'd) Model Generalization

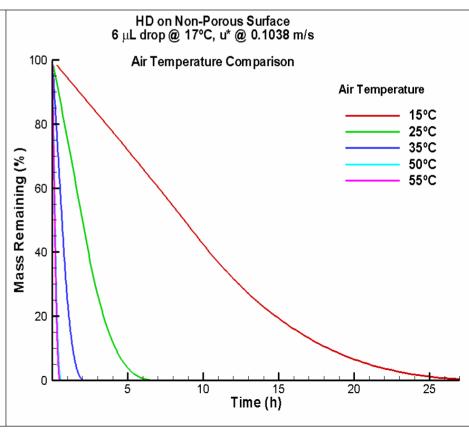
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Evaporation Module (cont'd) HD on a Non-Porous Surface – Parametric Studies

Effect of Velocity and Air Temperature on Evaporation







- All the scenarios presented in the matrix will be connected by two methods:
 - Classical curve-fit
 - √To have a simple engineering tool
 - ■Neural network
 - √To have a more sophisticated tool with prediction capabilities



Evaporation Module (cont'd)

Model Generalization

Caltech Work: Overview

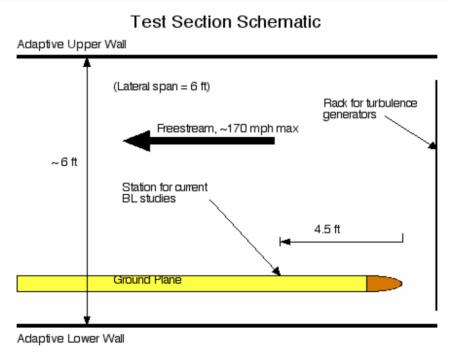
Motivating hypothesis (Navaz): The relevant velocity scale for fluid evaporation rate is not the freestream speed (U_{∞}) , but the "friction velocity"

$$u_{\tau} = \sqrt{\frac{\tau_{w}}{\rho}}$$

- > Variation of τ_w (and thus friction velocity) for given free stream speed demonstrated
- Simple case of drop evaporation rate evaluated experimentally
- Variation in evaporation rate with friction velocity observed
- Future experiments



John W. Lucas Adaptive Wall Wind Tunnel (GALCIT)



- > Turbulence generators (may be) placed in rack upstream of ground plane to increase freestream turbulence level.
- Boundary layer properties examined on ground plane at specified distance from leading edge.



Free Stream Turbulence Generation

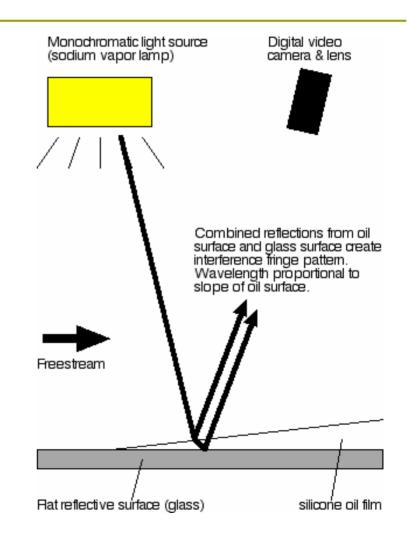


- Bungee cords stretched within frame in free stream.
- Act as turbulence generator; also vibrate quite a bit at high speeds.
- Various configurations available to "dial in" turbulence level.
- Idea: Bahram Valiferdowsi



Wall Shear Stress Measurement

- Oil film technique used by Nagib and others.
- Relatively nonintrusive - camera and lamp placed in tunnel, but near test section roof.
- By observing interference fringes growing with time, wall shear stress may be calculated.



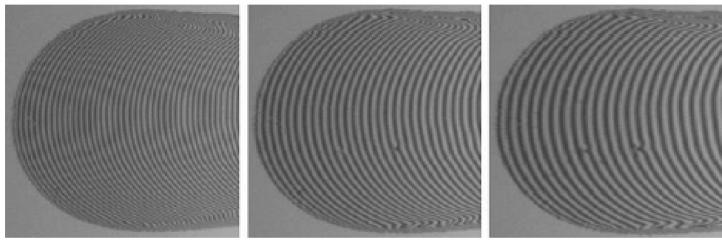


Shear Stress Measurement Apparatus





Wall Shear Stress Measurements



- Numerous images taken at 10-30 sec intervals.
- Fringe spacing growth rate, ds/dt, easy to evaluate.

$$\tau_{w} = \frac{2n\mu}{\lambda} \frac{ds}{dt}$$

 \triangleright n= oil index of refraction; $\mu=$ dynamic viscosity; $\lambda=$ light wavelength.



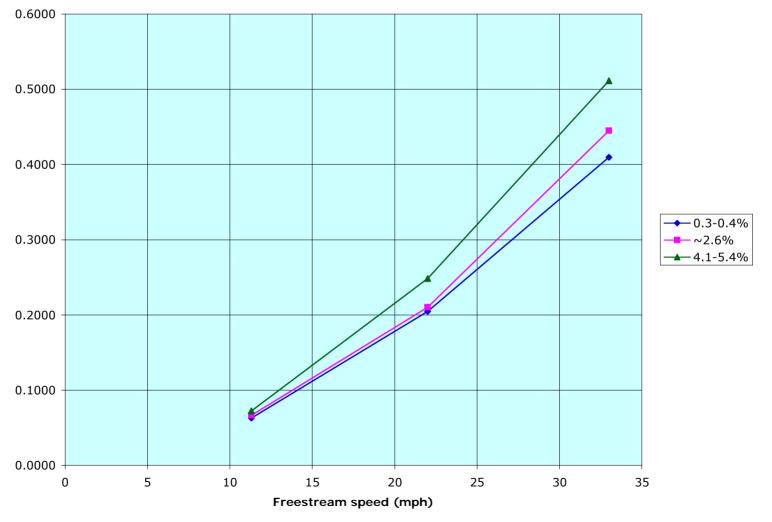
Wall Shear Stress Variation with Free Stream Turbulence

	Tu	rbulence Lev	vel
Speed (mph)	0.3-0.4%	~2.6%	4.1-5.4%
11.3	0.0626	0.0663	0.0723
22	0.2048	0.2102	0.2483
33	0.4096	0.4450	0.5115

- \triangleright Wall shear stress (τ_w) given above in Pa.
- \triangleright Notable increase in τ_w as turbulence level increases.
- Pata at 1% turbulence level showed slight decrease in τ_w ; calibration drift of pitotstatic pressure transducer suspected.

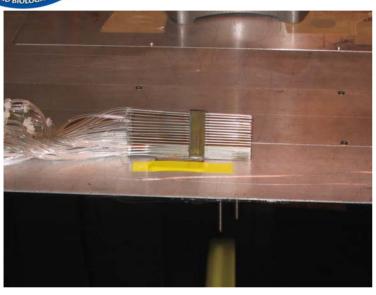


Free Stream Turbulence Intensity Affecting $\tau_{\rm w}$ - Plot





Boundary Layer Profile Measurement

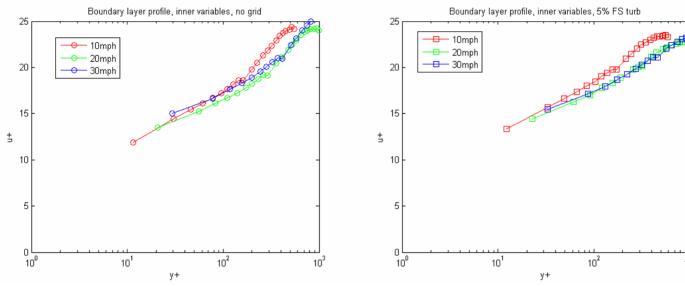




- 22-element boundary layer rake used to measure dynamic pressures at select heights
- Static pressure taken from nearby pitot-static tube



Boundary Layer Profile Comparison - ~0.4% vs. 5% free Stream Turbulence

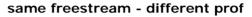


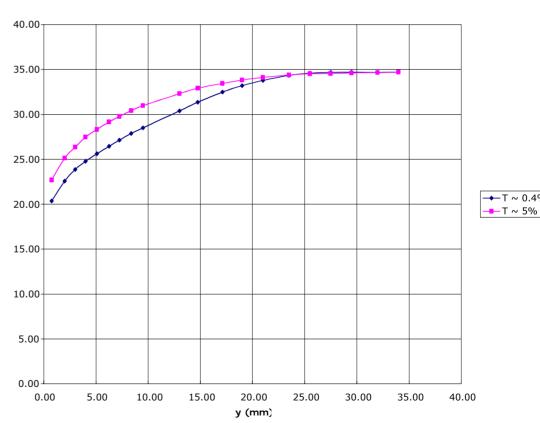
- Both profiles follow Clauser turbulent boundary layer profile; shape changes slightly near free stream transition
- \triangleright In no case is laminar sub-layer accessible (y+<5).



Profile Shape Change with Free Stream Turbulence Level

- At the same freestream speed, the shape of the boundary layer visibly varies with added freestream turbulence T.
- The higher T curve "stays high" lower, requiring a more abrupt reduction to zero.
- This agrees with the observed higher shear stress at the wall.





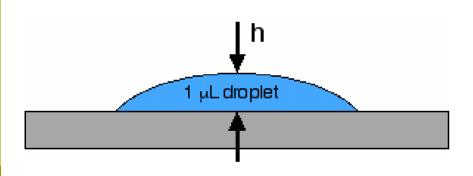


Critical Parameter for Evaporation: Wall Shear?

- It is hypothesized (Navaz) that evaporation rate of liquid droplets is based on the friction velocity (u_{τ}) rather than the free stream speed (U_{∞}) .
- \triangleright Current experiments demonstrate that for a constant U_{∞} , τ_{w} may be increased by up to 25% by imposing 5% turbulence intensity on the free stream flow.
- \gt 25% increase in $\tau_{\rm w}$ -> 12% increase in u_{τ} .
- Change in evaporation rate by up to 20% thus expected by model.



Relative Sizes: 1 µL Droplet and Laminar Sub-Layer



- 10 mph free stream and low turbulence level, present BL experiments
- Laminar sub-layer thickness (y+ = 5) is ~ 0.3 mm
- > 1 μ L droplet has height h of \sim 0.2 mm (Navaz)
- Droplet lies entirely within laminar sub-layer, where friction velocity u_τ is the dominant flow parameter (and only velocity scale!).



Proof-of-Concept Experiment

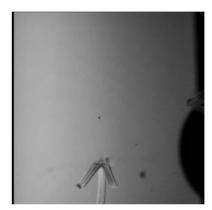
- ➤ Evaporation rate of 2.5 ml water drop in 10 mph free stream examined at turbulence intensity levels of 0.4% and 5%.
- Droplet dispensed on glass, video camera observes evaporation.
- Ensembles for each case taken, mean evaporation time recorded.



Droplet Evaporation Experiment







Turbulence intensity level (%)	0.4	5
Friction velocity (m/s)	0.23	0.25
Mean drop evaporation time (sec)	900	850



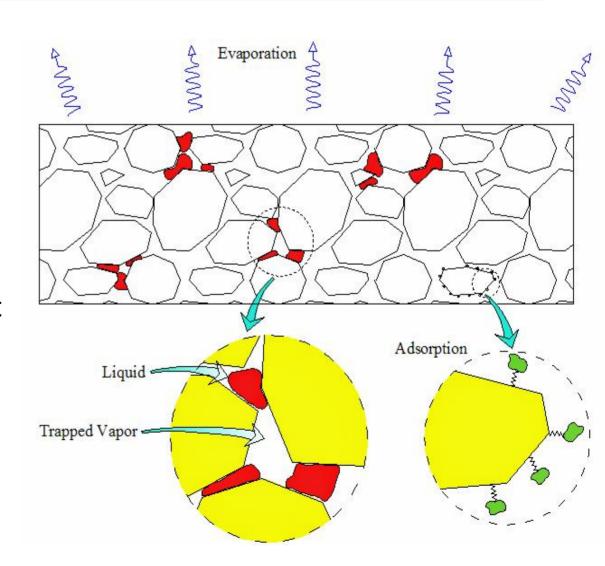
Upcoming Work at Caltech

- > Evaporation rate measurement
 - Instantaneous rate measurement
 - More accurate optical techniques
 - Mass measurement (microbalance?)
- Still higher turbulence intensity levels
- Further collaboration/verification with numerical model of H. Navaz
- Different surfaces (concrete, etc.)



Evaporation Module Link to the Porous Surface

- Simultaneous process present with evaporation
 - Capillary diffusion
 - Secondary evaporation
 - Vapor entrapment
 - Adsorption





Liquid simulants established a finite network in a porous domain (Czech concrete)





Porous Media/Substrate Modeling Proposed Approach

- Modeling effort
 - Solve the governing equation by capillary network model (CNM)
- Verification effort
- Generating the Design of Experiment matrix
- Obtaining solution for the entire matrix
- Curve fitting
 - Classical
 - Neural Network



Porous Media/Substrate Modeling Adding Adsorption Model

- Add adsorption model to the porous substrate model
- > Ensure robustness of the algorithm
- ➤ Validate model by
 - Conducting laboratory tests
- Verify the model
 - Laboratory
 - Outdoor



Conclusion

- Developed an evaporation model
- Verified the model with experimental data
- Extended the domain of applicability by using a hybrid analytical/experimental method
- Developed a framework for tackling a more complex problem – HD on porous substrate
- Resolving wind/turbulence/shear stress issues
- Incorporating the effects of wind turbulence intensity on evaporation



Evaporation Rate Validation

Evaporation of HD on Glass. 25 Drops/25 cm² Each Drop = 1 μ L - Wind Velocity = 5.8 ft/s

